

SST mechanical designs: preliminary evaluations for D-C and S-C options

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Presentation Outline

1. Davies-Cotton class Telescope

- a. General specifications
- b. Conceptual design
- c. FE Analyses:
 - a. Gravity effects
 - b. Wind effects
 - c. Eigenfrequencies search
- d. Final remarks

2. Schwarzschild-Couder class Telescope

- a. General specifications
- b. Conceptual design
- c. FE Analyses:
 - a. Model description
 - b. Eigenfrequencies search
- d. Final remarks

All the proposed concepts have to be considered preliminary. Designs could be subjected to changes and revisions



	D-C 7m	S-C 4m	S-C 7m	3-MT 4m	3-MT 7m
Focal length	10500 mm	2000 mm	3500 mm	2000 mm	3500 mm
Lever arm	10500+det	2300 mm	4000 mm	600 mm	1050 mm
θ _{ΕΕ(90)}	2x3°	2x2.5°	2x2.5°	2x5°	2x5°
Detector	Single-pixel 36 mm	Multi-pixel 7 mm	Single-pixel 12 mm	Multi-pixel 7 mm	Single-pixel 12 mm
PMT units	1200	75	1200	75	1200
1-vignetting	90% 1-refl	70% 2-refl	70% 2-refl	70% 3-refl	70% 3-refl
Total throughput	77%	51%	51%	43%	43%
Structure cost	10	1	4	1	4
Array (tel. number)	20	100	30	118	35
Mirror sags (1x1m) - M1 - M2	12 mm	25 mm 230 mm	14 mm 130 mm	40 mm 92 mm 160/80 mm	46 mm 106 mm 184/92 mm
- M3					104/92 11111



Main general specifications

Ø7m class telescope

Single mirror f=10.5m

Mirror curvature radius: 21m

Camera mass: 1.6tons

Field of view: 8°

Operating Wind: 50Km/h

Dish contribution to PSF* < 1mrad Camera displacement 0.5PMT pixel

10.5m

*PSF being computed as the RMS of ray deviation at focal plane



Conceptual Design

Main features

Alt-Azimuthal mounting.

Axial and lateral loads transmitted to the concrete foundations through pintle bearing.

Elevation lattice mast structure with tendons interfaced with steel azimuth fork

Ballast boxes envisaged as counterweights to grant correct balancing around elevation axis.









Conceptual Design

Primary Mirror

• 36 Hexagonal shaped sandwich mirror panels



 $S \approx 1.2m$

(TBC)

- Positioning of each sector activecontrolled through 2 actuators + 1 fixed point mounted on a "tripod" also accommodating the interface with the structure or an intermediate structure layer accommodating panel supports
- Lattice supporting dish





Conceptual Design



• Elevation motion performed through a worm drive



• Azimuthal positioning controlled through gears and drives on the pintle bearing (accessible from foundation room).



Mirror actuators



2 actuators + 1 fixed point for tip/tilt movement.

Fixed point:

- orthogonal foils to give flexibility along x and y (for tilt) and rigidity along z

Actuators:

- linear actuator with gearmotor;
- elongated orthogonal foils block mirror translation without inserting additional loads;
- relative positioning (no encoder) to save money





FE Analyses: Gravity effects

Telescope FEM implemented; Analyses performed in order to evaluate deflections related to gravity effects





FE Analyses: Wind effects





- 1. Mirror panels FEM implemented
- 2. Entity of wind pressure on the mirror evaluated through proper analyses
- 3. Forces and moments resulting from integration of pressures onto single sectors applied to whole telescope FEM
- 4. Deflections related to wind effects onto the panels and mast predicted



FE Analyses: Prediction of optical degradation at camera focal plane

Displacements and rotations of mirror panels and camera retrieved (Gravity & Wind effects) Rough estimation of image at focal plane performed, RMS computed





FE Analyses: Eigenfrequencies search





Final remarks

General design

- 1. Simple and basic design.
- 2. Simple Mirror with sandwich panel sectors
- 3. Altitude pointing performed through worm drive assembly
- 4. Azimuthal pointing performed through gears and drives

Preliminary FE Analyses

- 1. Panel and camera displacements/rotations due to gravity and wind leading to a PSF < 1mrad
- 2. First eigenfrequency above 2.5Hz : general design acceptable but still to be improved; 3Hz goal accessible



Main general specifications

- Ø4m class telescope
- Two-mirror telescope
- M1 curvature radius: ~10m
- M2 curvature radius: ~2.5m
- Camera mass: 100 Kg
- Operating Wind: 50Km/h
- Contribution to PSF < 1mrad Camera displacement 0.5PMT pixel









Conceptual Design

Main features

- "Cardan" mount, pointing performed through worm drive system only: Allowed elevations: 30° to 90°
 Azimuth pointing: -180° to 180°
- 2. Axial and lateral loads transmitted to the concrete foundations through universal altitude joint and worm-drives
- 3. Simple elevation mast structure supporting M2 and Camera
- 4. Counterweights envisaged to grant correct balancing around universal joint.





Conceptual Design (alternatives)





Conceptual Design

In the meantime, a "classical" configuration with alt-azimuthal mounting is at present under investigation.

Costs and performances trade-off between the different solutions will be performed.





"Cardan" vs Alt-Az

"Car	dan"	Alt~Az		
Advantages	Disadvantages	Advantages	Disadvantages	
Lightweight and "essential"	Pedestal could be quite high or with a fashioned design	Well known and standard solution	Higher costs for both structure and motors	
More accurate measure of the dish positioning (measured directly on the dish)	Pointing limited to >25° from horizon, mainly due to eigenfrequencies	Secondary can be lowered close to ground	Less accurate pointing and tracking (measured on gears and drives)	
No tracking limits			Tracking limit at zenith	
On-site assembly more easy and fast				



























Conceptual Design

Primary mirror (M1)

• 18+(1) hexagonal/petal shaped sandwich mirror panels



 $S\approx 0.85m$

TBC



- Positioning of each sector activecontrolled through 3 actuators mounted on a "plate" also accommodating the interface with the structure.
- Lattice supporting dish



Conceptual Design

Secondary mirror (M2)

- 6 sandwich mirror sectors
- Positioning of each sector active-controlled through 3 actuators mounted on a "plate" also accommodating the interface with the structure





Preliminary FE Analyses: Model description

Numerical model of M1dish, mast, M2 and camera implemented. Mirror panels and camera lumped and modeled as simple structural masses.

Concrete foundation considered as infinitely stiff.





Preliminary FE Analyses: Eigenfrequencies search (Elevation = 30°, Azimuth = 0°)





Mode 1: ~ 5.6Hz Mast global torsion Eigen-frequency mainly related to mast stiffness

Mode 2: ~ 5.7Hz Global rotation about the elevation universal joint Eigen-frequency mainly related to worm-drives stiffness



Preliminary FE Analyses: Eigenfrequencies search (Elevation = 30°, Azimuth = 180°)



Mode 1: ~4.0Hz Global rotation about elevation universal joint Eigen-frequency mainly related to worm-drives stiffness Mode 2: ~5.0Hz Mast global bending Eigen-frequency mainly related to dish stiffness



Final remarks

General design

- 1. Innovative design.
- 2. Altitude and azimuthal pointing performed through worm drive system only.
- 3. Significant saving of structure mass, components number \rightarrow global cost
- 4. "Classical" configuration with alt-az mounting under investigation. Costs and performances Trade-off between the two solutions will be performed.

Preliminary analyses

- 1. Panel and camera displacements/rotations due to gravity and wind leading to a PSF < 1mrad
- 2. First eigenfrequency above 4Hz : Value acceptable even in case of reduction when additional flexibility of joints and mechanical parts is taken into account



Final remarks

- Two design under investigation
- Both satisfy the main requirements





- Trade-off study is necessary with a global approach (not only structure)

- Need to start a feasibility study